Modelling of Turbulent Flows

Lecture 1.2

March 24, 2020

Vorticity Dynamics

Vorticity Dynamics

Definition of Vorticity

Vorticity Transport Equation (for Incompressible Fluids)

Vorticity Stretching

Baroclinic Effect

Kelvin's Theorem

When we consider the flow of a fluid (with density ρ and viscosity μ) past a bluff body we can analyze the flow field in the following way:



We can identify three regions:



1 Potential flow Far from the body but with deformation of the streamlines:

Negligible viscous dissipation $Vorticity = 0 \label{eq:viscous}$



2 Wake region characterized by vortex stretching:

Negligible viscous dissipation

Non-zero vorticity



3 Wall region Boundary layer:

Important viscous dissipation Non-zero vorticity

The velocity gradient near the wall is important due to the no-slip boundary dissipation due to the viscous stress:

$$\tau_{ij} = \mu \left(\frac{\partial v_i}{\partial x_j} + \frac{\partial v_j}{\partial x_i} \right)$$

Definition of vorticity

$$\vec{\omega} = \operatorname{rot} \vec{v} = \vec{\nabla} \times \vec{v} = \begin{vmatrix} \hat{i} & \hat{j} & \hat{k} \\ \frac{\partial}{\partial x} & \frac{\partial}{\partial y} & \frac{\partial}{\partial z} \\ v_x & v_y & v_z \end{vmatrix}$$
$$= \vec{i} \left(\frac{\partial v_z}{\partial y} - \frac{\partial v_y}{\partial z} \right) + \vec{j} \left(\frac{\partial v_x}{\partial z} - \frac{\partial v_z}{\partial x} \right) + \vec{k} \left(\frac{\partial v_y}{\partial x} - \frac{\partial v_x}{\partial y} \right)$$

Vorticity represents the local rotation rate of an elementary parcel of fluid. Since vorticity is defined by the derivatives of the velocity vector, it is also related to the deformation rate.

We can now try to make some examples to understand better the role of vorticity.

A fluid is rotating as if it were a rigid body with angular rotation rate Ω . The velocity field is

$$\vec{u} = \vec{\Omega} \times \vec{r}$$

with $\vec{\Omega} = \Omega_z \hat{k}$ and $\vec{r} = x\vec{i} + y\vec{j}$.

$$\vec{v} = \Omega_z \hat{k} \times (x\hat{i} + j\hat{j}) = \begin{vmatrix} \hat{i} & \hat{j} & \hat{k} \\ 0 & 0 & \Omega_z \\ x & y & 0 \end{vmatrix} = \Omega_z \cdot x\hat{j} - \Omega_z \cdot y\hat{i}$$

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and $v_z = 0$.

The components of vorticity are:

$$\omega_{x} = \frac{\partial v_{z}}{\partial y} - \frac{\partial v_{y}}{\partial z} = -\frac{\partial}{\partial z}(\Omega_{z} \cdot x) = 0$$

$$\omega_{y} = \frac{\partial v_{x}}{\partial z} - \frac{\partial v_{z}}{\partial x} = \frac{\partial}{\partial z}(\Omega_{z} \cdot y) = 0$$

$$\omega_{z} = \frac{\partial v_{y}}{\partial x} - \frac{\partial v_{x}}{\partial y} = \frac{\partial}{\partial x}(-\Omega_{z} \cdot y) - \frac{\partial}{\partial y}(\Omega_{z} \cdot x) = 2\Omega_{z}$$

 $\Rightarrow \vec{\omega} = \omega_z \hat{k}$ orthogonal to the motion plane.

Example 2

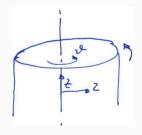
Every fluid particle is moving on a circular path about the z-axis, but with the radial velocity distribution corresponding to the torsional flow.

 $v_{\theta} = \frac{k}{r}$ with k = constant. It is the flow generated in a cylindrical container by the boundary which moves at constant speed. Radial and azimuthal vorticities are null:

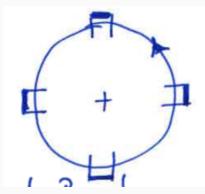
$$\omega_r = \frac{1}{r} \frac{\partial v_z}{\partial \theta} - \frac{\partial v_\theta}{\partial z} = 0$$
$$\omega_\theta = \frac{\partial v_\theta}{\partial z} - \frac{\partial v_z}{\partial z} = 0$$

while

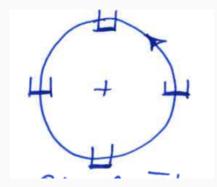
$$\omega_{z} = \frac{1}{r} \frac{\partial}{\partial r} (rv_{z}) - \frac{1}{r} \frac{\partial v_{r}}{\partial \theta} = \frac{1}{r} \frac{\partial}{\partial r} (r \cdot \frac{k}{r}) = 0$$



In the limit of \vdash very small:

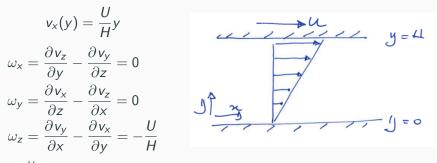


Rigid body rotation



Circulation without rotation

This is the simple shear flow in a Couette device..



 $\frac{U}{H}$ is, of course, the slope of the velocity profile.

Plane Poiseuille flow.

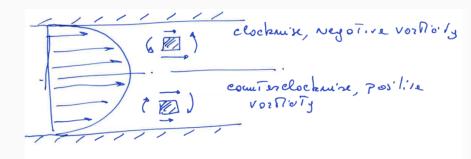
$$v_{x}(y) = \frac{1}{2\mu} \left(\frac{\Delta P}{L}\right) \left[y^{2} - \frac{H^{2}}{4}\right]$$

$$\omega_{x} = \omega_{y} = 0$$

$$\omega_{z} = -\frac{\partial v_{x}}{\partial y} = -\frac{1}{2\mu} \left(\frac{\Delta P}{L}\right) 2y$$

$$= -\frac{1}{\mu} \frac{\Delta P}{L} y$$

Maximum vorticity (magnitude) is at both walls. Vorticity is zero in the centerplane.



The vorticity transport equation describes the space and time evolution of vorticity. The equation is obtained by applying the curl operation to all terms of the Navier-Stokes equation.

$$\operatorname{rot}\left[\rho\left(\frac{\partial \vec{v}_{l}}{\partial t} + \vec{v} \cdot \vec{\nabla} \vec{v}\right)\right] = \operatorname{rot}\left[-\vec{\nabla}P + \mu \nabla^{2} \vec{v}\right]$$
$$\rho\left(\vec{\nabla} \times \frac{\partial \vec{v}}{\partial t} + \vec{\nabla} \times (\vec{v} \cdot \vec{\nabla} \vec{v})\right) = -\vec{\nabla} \times \vec{\nabla}P + \vec{\nabla} \times (\mu \nabla^{2} \vec{v})$$
$$\frac{\partial}{\partial t}\left(\vec{\nabla} \times \vec{v}\right) + \vec{\nabla} \times (\vec{v} \cdot \vec{\nabla} \vec{v}) = -\frac{\vec{\nabla} \times \vec{\nabla}P}{\rho} + \frac{\mu}{\rho} \vec{\nabla} \times (\nabla^{2} \vec{v})$$
vorticity

Some useful vector properties

I.
$$\forall$$
 scalar field $A\vec{\nabla} \times \vec{\nabla}A = 0 \Rightarrow \vec{\nabla} \times \vec{\nabla}P = 0$
II. $\vec{\nabla} \times \vec{\nabla}\vec{v} = \nabla^{2}\vec{\omega}$
III. $\vec{\nabla} \times (\vec{v} \cdot \vec{\nabla}\vec{v}) = (\vec{v} \cdot \vec{\nabla})\vec{\omega} - (\vec{\omega} \cdot \vec{\nabla})\vec{v}$
 $= 0 \text{ in } 2D$

because

$$\vec{\nabla} \times (\vec{v} \cdot \vec{\nabla} \vec{v}) = \vec{\nabla} \times \left[\vec{\nabla} \left(\frac{1}{2} \vec{v} \cdot \vec{v} - \vec{v} \times \vec{\omega} \right) \right]$$
$$= \vec{\nabla} \times \left[\vec{\nabla} \left(\frac{1}{2} \vec{v}^2 \right) \right] - \vec{\nabla} \times (\vec{v} \times \vec{\omega})$$
$$= -\vec{v} (\vec{\nabla} \cdot \vec{\omega}) + \vec{\omega} \underbrace{(\vec{\nabla} \cdot \vec{v})}_{\text{incompr.}} + (\vec{v} \cdot \vec{\nabla}) \vec{\omega} - (\vec{\omega} \cdot \vec{\nabla}) \vec{v}$$
$$= -\vec{v} \underbrace{\vec{\nabla} \cdot (\vec{\nabla} \times \vec{v})}_{\text{incompr.}} + (\vec{v} \cdot \vec{\nabla}) \vec{\omega} - (\vec{\omega} \cdot \vec{\nabla}) \vec{v} = (\vec{v} \cdot \vec{\nabla}) \vec{\omega} - (\vec{\omega} \cdot \vec{\nabla}) \vec{v}$$
$$= 0 \text{ in } 2D$$

The equation becomes

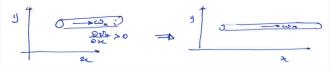


where *Re* is obtained upon proper non-dimensionalization of the equation.

Considering for simplicity just one component (x), the vortex stretching term is

$$\left[\left(\vec{\omega} \cdot \vec{\nabla} \vec{v} \right)_{x} = \underbrace{\omega_{x} \frac{\partial v_{x}}{\partial x}}_{\text{vortex stretching part}} + \omega_{y} \frac{\partial v_{x}}{\partial y} + \omega_{z} \frac{\partial v_{x}}{\partial z} \right]$$

The vortex stretching part acts when a velocity gradient exists in the same direction of vorticity.



Due to this action, when the fluid parcel is stretched, then, to conserve the angular momentum, there will be a corresponding rotation rate increase and consequently an increase of vorticity (much like the rotation speed of an ice-skate dancer). N.B. This effect is very important in turbulence because it **helps** creating smaller scales.

This effect is an *auto-amplification* effect: just due to the alignment of velocity gradients and vorticity there is an increase in vorticity.

Considering for simplicity just one component (x), the vortex stretching term is

$$\left[\left(\vec{\omega} \cdot \vec{\nabla} \vec{v} \right]_x = \omega_x \frac{\partial v_x}{\partial x} + \omega_y \frac{\partial v_x}{\partial y} + \omega_z \frac{\partial v_x}{\partial z} \right]_x$$
vorticity transfer

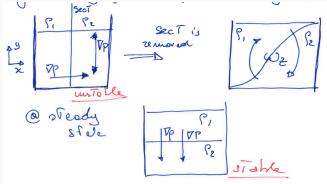
The other two terms, $\omega_y \frac{\partial v_x}{\partial y}$ and $\omega_z \frac{\partial v_x}{\partial z}$, contribute to *rotate* part of the existing vorticity and therefore to **transfer vorticity** from one component to the other.

We have considered an incompressible fluid. If we allow difference in density we might have density gradients and the pressure term does not disappear from the vorticity equation, which becomes:

$$\frac{\partial \vec{\omega}}{\partial t} + (\vec{v} \cdot \vec{\nabla}) \vec{\omega} = \frac{\vec{\nabla} \rho \times \vec{\nabla} P}{\rho^2} + \nu \nabla^2 \vec{\omega} + (\vec{\omega} \cdot \vec{\nabla}) \vec{v}$$

Usually, when density is allowed to vary, the density gradient is aligned with the pressure gradient (ocean density, atmospheric density in stable conditions). However, situations may arise when these two gradients are not aligned and vorticity is produced. Consider the following example:

In a container we have low and high density fluid separated by a sect.



In a steady, 2D incompressible flow the vorticity equation is

$$rac{\partial ec \omega}{\partial t} = 0 \Rightarrow (ec v \cdot ec
abla) ec \omega = rac{1}{Re}
abla^2 ec \omega$$

Assuming negligible viscous dissipation, we have $\nu \to 0 \Rightarrow Re \to \infty \text{ and}$

$$(ec{v}\cdotec{
abla})ec{\omega}=0$$

In 2D $\vec{\omega} = \omega$ and $\vec{v} \cdot \vec{\nabla} \omega = 0$, which implies that $\vec{v} \perp \vec{\nabla} \omega$.

Kelvin's theorem states that in an inviscid fluid ($\nu = 0$) the circulation of a material tube is constant. The circulation is

$$\Gamma = \int \omega \mathrm{d}S$$

where dS is the differential surface of a material tube.

If we are far from the body and the fluid is *irrotational* (i.e. $\omega = 0$) in that region, Kelvin's theorem states that the flow field is irrotational everywhere.